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## STATIC MODELING OF A BI-AXIAL MICRO-MIRROR WITH SIDEWALL ELECTRODES

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### ABSTRACT

*The static modeling of a bi-axial torsional micro-mirror with sidewall electrodes is investigated. The mirror and sidewall electrodes experience different voltages. As a result of the potential difference, the mirror rotates about x and y axes with the torques generated from electrostatic forces by the sidewall and bottom electrodes. The rotations about two axes are made possible using a gimbal frame and serpentine springs. An analytical model is presented that is a simplified model of a previous study based on independent excitation of the two rotation angles. The simplified model enables prediction of the rotation angles with good accuracy and faster computation time.*

### 1 INTRODUCTION

Micro-electromechanical systems (MEMS) are becoming mainstream using recent microfabrication methods (e.g Silicon bulk micro-machining, surface micro-machining [1]). Biaxial micro-mirrors are among MEMS devices adopted in numerous areas such as laser imaging, image digitizing, projection displays [2] and medical applications, e.g. endoscopy and tomography. Features to be mostly considered for micromirrors include simple fabrication processes, low driving voltage, large tilt angle and linearized angular scan [3]. Actuation methods used for micromirrors include electrostatic [4–10], piezoelectric [11–17](PZT), electrothermal [18–20] and electromagnetic [21–24] actuators. Electrostatic actuation is the most popular actuation methods due to easy fabrication, low power consumption

and high driving speed [25]. However, they require large operation voltages to have large tilting angles. Also for the MEMS devices that use parallel plate capacitors, pull in instability is a big concern [11, 26]. Electrothermal actuation mechanism provides large deflection angle for low voltage values, but their basic disadvantages are low driving speed and large energy consumption in addition to the change of optical behavior due to temperature [27, 28]. Piezoelectric actuators have small device sizes, better response of driving speed compared to other actuators and low driving voltages [14–17]. However, they have the disadvantage of difficult fabrication. Electromagnetics actuators can achieve large mechanical tilting angle [23, 29] but at the price of larger size. Among these actuation types electrostatic actuator is preferred due to ease of fabrication in large volumes. Common electrostatic actuator types are comb-drive and parallel plate. Parallel plate actuators are preferred due to their smaller size compared to comb-drive actuators.

In this study an analytical model is investigated for a bi-axial micro-mirror. The model was introduced by Bai et al. [30], who developed an electrostatic micro-mirror with sidewall electrodes. They have presented simulations and experimental results. Contribution of this study is to simplify the analytical model presented [30] based on the independent excitation of the two rotation angles. we also changed the boundaries of the integral to account for the changes in the forces that become significant as the rotation angles increase.

The sidewall electrodes are used to decrease voltages and to increase rotation angles. The reason for adding sidewall electrodes is to increase the area of electrostatic force according to

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the electrostatic force equation  $F = \frac{\epsilon_0 \cdot A \cdot V^2}{d^2}$ . Increasing forces on the mirror leads to larger torque around its axis of rotation and consequently larger rotation angle for the same amount of voltage. The micro-mirror structure consists of five main components; mirror plate, gimbal frame, sidewall electrode, bottom electrode and serpentine spring (torsion bar). The mirror is made of silicon and is connected to the ground and the voltage is applied to the sidewall and the bottom electrodes. Thus, the mirror is actuated by electrostatic forces generated from different potentials between the sidewall, the bottom electrodes and the mirror plate.

## 2 OPERATION PRINCIPLES

A schematic of the micro-mirror is shown in Figure 1. The micro-mirror is suspended by the double-gimbal structure, which has two pairs of serpentine springs (Figure 2) for 2 degree-of-freedom (DOF) scans:  $\alpha$  and  $\beta$  scan which are about x-axis and y-axis, respectively. The mirror actuators consist of four equivalent electrode quadrants (also sidewall electrodes) numbered as in Figure 1. The parameters of the micro-mirror are listed in Table 1. Different voltage potential between the mirror plate and the sidewall and bottom electrodes creates electrostatic torques that rotate the mirror. To operate the mirror, the two angles are excited independently. Voltages are applied to the sidewall and bottom electrodes of each quadrant according to the differential drive method suggested by Hao et al. [31] For the rotation about Y-axis ( $\beta$ )

$$\begin{aligned} V_1 &= V_2 = V_{bias} + V_{dif} \\ V_3 &= V_4 = V_{bias} - V_{dif} \end{aligned} \quad (1)$$

where  $V_{bias}$  is the bias voltage and  $V_{dif}$  is the differential voltage, and  $V_{1...4}$  is the voltage on each quadrant. For the rotation about X-axis ( $\alpha$ )

$$\begin{aligned} V_1 &= V_3 = V_{bias} - V_{dif} \\ V_2 &= V_4 = V_{bias} + V_{dif} \end{aligned} \quad (2)$$

In other words, quadrants Q1 and Q2 will have equal and larger voltage compared to other quadrants to make the rotation angle  $\beta$ . Similarly, quadrants Q2 and Q4 will have equal and larger voltage compared to other quadrants to make the rotation angle  $\alpha$ .

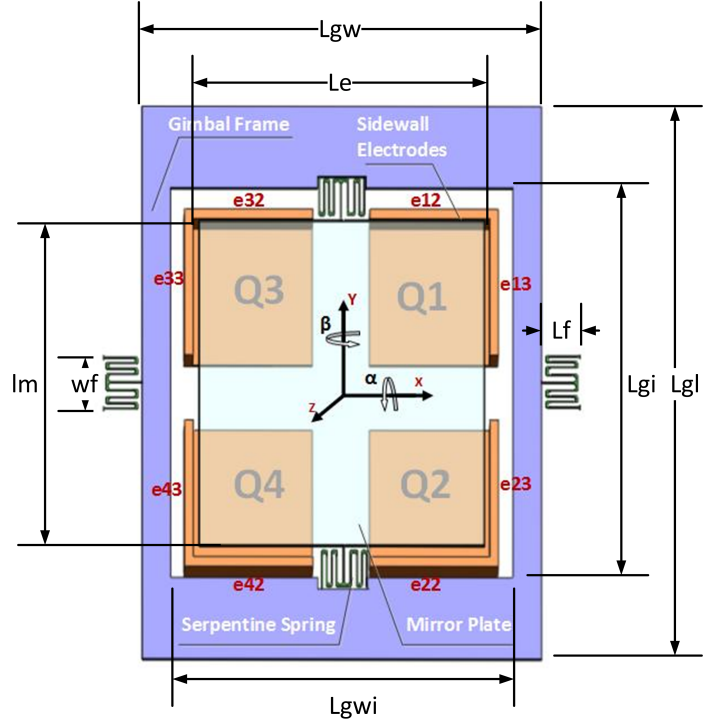


FIGURE 1: Mirror Elements and Quadrants

FIGURE 2: Serpentine Spring



## 3 SYSTEM MODEL

### 3.1 Electrostatic Forces and Torques

The global coordinate is XYZ and fixed at the center of the initial position of mirror center. The body fixed coordinate xyz is fixed at the center of the mirror plate and is rotating with the micro-mirror. Rotations about x and y axes are done independently. Using the defined coordinate systems, equations for forces and torques on the mirror from sidewall electrodes, gimbal frame and bottom electrodes are obtained, respectively in this section for angle  $\alpha$ . The derivation for angle  $\beta$  is presented in the Appendix.

**3.1.1 Sidewall electrodes** Figure 3 shows the projection of the mirror plate on the YZ plane. The center of the mirror plate passes through the center of the global coordinate. The figure shows the mirror rotation about the X axis when quadrants 2 and 4 have equal and higher voltage than quadrants 1 and 3. In Figure 3, since the torques about the Y axis balance each other in this case, there will not be any rotation about the Y axis, so  $\beta = 0$ .

The model for electrostatic forces and torque here follow the procedure explained in Bai et. al [30], which consider the electrostatic forces from the sidewall 22 and 12 on the bottom surface of the mirror. However, we simplified the model based on the operation principle in the previous section (the two angles are operated independently). We also considered the electrostatic forces from the sidewall electrode on the top surface of the mirror as they become significant in large rotation angles. To incorporate this effect at large angles, the integral boundaries are defined that depend on the rotation angles obtained in the previous voltage step of the simulations.

For an element of  $dy$  on the top layer of the mirror plate, the electrostatic flux from sidewall 22 on the top surface of the mirror is written as;

$$E_{e22t} = \frac{V_2}{\widehat{AB}} \quad (3)$$

where  $V_2$  is the voltage applied on quadrant 2 and

$$\begin{aligned} \widehat{AB} &= AC \cdot (\pi/2 - \alpha) = \frac{AD}{\sin(\pi/2 - \alpha)} (\pi/2 - \alpha) \\ &= \left(\frac{L_e}{2} - Y\right) \frac{\pi/2 - \alpha}{\sin(\pi/2 - \alpha)} \end{aligned} \quad (4)$$

Substituting  $Y = y \cos \alpha$  for a body fixed point A, the flux equation 3 is written as;

$$E_{e22t} = \frac{V_2 \sin(\pi/2 - \alpha)}{\left(\frac{L_e}{2} - y \cdot \cos \alpha\right) (\pi/2 - \alpha)} \quad (5)$$

The electrostatic pressure is

$$P = \frac{\epsilon_0 E^2}{2} \quad (6)$$

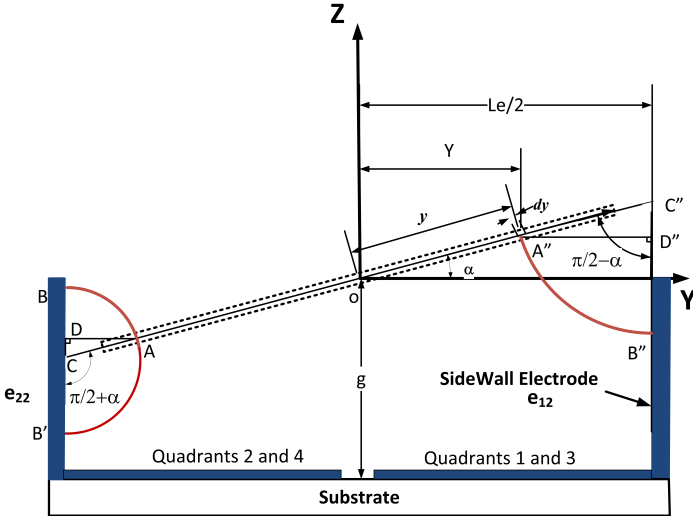
Therefore the electrostatic force on on element with dimensions of  $dx$  and  $dy$  is

$$dF = \frac{\epsilon_0 E^2 dx dy}{2} \quad (7)$$

**TABLE 1:** Parameters for micro-mirror structure [30] in Figures 1 and 2

Parameter	Symbol	Value
<b>Mirror Plate</b>		
Mirror width	lm	1000 $\mu$ m
Mirror length	lm	1000 $\mu$ m
<b>Gimbal Frame</b>		
Outer width	Lgw	1400 $\mu$ m
Inner width	Lgwi	1200 $\mu$ m
Outer length	Lgl	1700 $\mu$ m
Inner length	Lgi	1200 $\mu$ m
Thickness	tg	35 $\mu$ m
Width in Y direction	$g_{wy}$	250 $\mu$ m
<b>Serpentine Spring</b>		
Width	w	1.5-3 $\mu$ m
Thickness	$t_b$	12 $\mu$ m
$l_0$		10 $\mu$ m
$l_p$		110 $\mu$ m
$l_f$		120 $\mu$ m
$l_i$		120 $\mu$ m
$w_f$		160 $\mu$ m
<b>Electrode Dimensions</b>		
Bottom edge width	Le	1060 $\mu$ m
Gap	g	290 $\mu$ m
Sidewall thickness	$t_s$	40 $\mu$ m
Sidewall height	h	250 $\mu$ m
<b>Material Properties</b>		
Silicon density	$\rho$	2329 $kg/m^3$
Young's Modulus	E	79 Gpa
Shear Modulus	G	73 Gpa
Air permittivity	$\epsilon_0$	$8.854e - 12 \frac{F}{m}$

The force in the Z direction from sidewall  $e22$  is then found



**FIGURE 3:** The mirror rotated about X axis by  $\alpha$ , and the sidewalls 12 and 22 projected on the YZ plane.

from integration

$$S_1 = \frac{\epsilon_0 \cdot \sin^2(\pi/2 - \alpha)}{2} \quad (8)$$

$$F_{z_{e22t}} = \frac{S_1 V_2^2}{(\pi/2 - \alpha)^2} \cdot \int_{x_1}^{x_2} \int_{y_1}^{y_2} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot dx dy$$

where  $y_1 = \frac{L_e}{2 \cos(\alpha_0)} - \frac{L_e}{2} \tan(\alpha_0)$ ,  $y_2 = \frac{lm}{2}$ ,  $x_1 = \frac{wf}{2}$  and  $x_2 = \frac{lm}{2}$ , and  $\alpha_0$  is the rotation angle in the previous voltage step. In other words, the boundaries of the integral changes as the rotation angle changes. At small angles, that does not make a difference in the results, but as the rotation angles grow, this effect becomes more significant.

From the force equation, one can find the torque about the x axis from sidewall  $e22$  on top surface of the mirror plate.

$$T_{x_{e22t}} = - \frac{S_1 V_2^2}{(\pi/2 - \alpha)^2} \cdot \int_{x_1}^{x_2} \int_{y_1}^{y_2} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot y dx dy \quad (9)$$

For an element of  $dy$  at the bottom surface of the mirror plate (3), the electrostatic flux from sidewall  $e22$  is written as;

$$E_{e22b} = \frac{V_2}{\widehat{AB'}} \quad (10)$$

where

$$\widehat{AB'} = AC \cdot (\pi/2 + \alpha) = \frac{AD}{\sin(\pi/2 - \alpha)} (\pi/2 + \alpha) \quad (11)$$

$$= \left( \frac{L_e}{2} - Y \right) \frac{\pi/2 + \alpha}{\sin(\pi/2 - \alpha)}$$

Substituting  $Y = y \cos \alpha$ , the flux equation (10) is written as;

$$E_{e22b} = \frac{V_2 \sin(\pi/2 - \alpha)}{\left( \frac{L_e}{2} - y \cdot \cos \alpha \right) (\pi/2 + \alpha)} \quad (12)$$

The force in the Z direction from sidewall  $e22$  on the bottom part of the mirror is

$$F_{z_{e22b}} = - \frac{S_1 V_2^2}{(\pi/2 + \alpha)^2} \cdot \int_{x_3}^{x_4} \int_{y_3}^{y_4} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot dx dy \quad (13)$$

The boundaries of the integral can be found in Appendix B. The torque about the x axis from sidewall  $e22$  at the bottom of the mirror plate is then

$$T_{x_{e22b}} = \frac{S_1 V_2^2}{(\pi/2 + \alpha)^2} \cdot \int_{x_3}^{x_4} \int_{y_3}^{y_4} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot y dx dy \quad (14)$$

Electrostatic forces from sidewall 12 also changes the rotation angle  $\alpha$ . As it can be seen from Figure 3, the electrostatic flux on the bottom surface of the mirror plate from side wall 12 is

$$E_{e12} = \frac{V_1}{\widehat{A''B''}} = \frac{V_3}{A''C'' \cdot (\pi/2 - \alpha)} \quad (15)$$

where  $V_1$  is the voltage of quadrant 1. The force and torque on the mirror plate from sidewall 12 can be derived similarly to equation (9) with different boundary for the integral.

$$F_{z_{e12}} = - \frac{S_1 V_1^2}{(\pi/2 - \alpha)^2} \cdot \int_{x_5}^{x_6} \int_{y_5}^{y_6} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot dx dy \quad (16)$$

$$T_{x_{e12}} = - \frac{S_1 V_1^2}{(\pi/2 - \alpha)^2} \cdot \int_{x_5}^{x_6} \int_{y_5}^{y_6} \left\{ \frac{1}{\frac{L_e}{2} - y \cdot \cos \alpha} \right\}^2 \cdot y dx dy$$

Now we consider the effect of electrostatic forces from the sidewall electrodes on the gimbal frame which contributes to the rotation of the mirror. The gimbal frame does not rotate about Y

axis based on Figure 4, and it can only rotate about X axis. That means the electrostatic forces generated between sidewalls and gimbal frame changes  $\alpha$  angle only. Figure 4 shows the projection of the mirror plate and the gimbal frame on a plane parallel to YZ plane. Due to larger width of the gimbal frame in the y direction, only the electrostatic torques and forces caused by the sidewalls 12,22,32 and 42 are considered in the simulations.

The resulting electrostatic force in the Z direction and torque around the X axis caused by sidewall 12 forces on the top surface of the gimbal can be written:

$$\begin{aligned} S_2 &= \frac{\epsilon_0 \cdot \sin^2(\pi/2 - \alpha)}{2} \\ Fz_{eg22t} &= \frac{S_2 V_2^2}{(\pi/2 + \alpha)^2} \int_{x_7}^{x_8} \int_{y_7}^{y_8} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .dxdy \\ Tx_{eg22t} &= -\frac{S_2 V_2^2}{(\pi/2 + \alpha)^2} \int_{x_7}^{x_8} \int_{y_7}^{y_8} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .ydxdy \end{aligned} \quad (17)$$

It should be noted that the above equations are only valid when the angle of  $\alpha$  is large enough so that integral boundaries  $y_8 = Le/(2 \cdot \cos \alpha_0) + Le/2 \cdot \tan \alpha_0 > y_7 = Lgl/2 - g_{wy}$  and that happens when  $\alpha_0 > 0.06$  radians. In other words, the forces on the top layer of gimbal frame only applies when the rotation angle  $\alpha$  reaches certain value of 0.06 radians.

The sidewall 22 also exerts electrostatic forces on the bottom surface of the gimbal frame. The corresponding electrostatic force and torque are

$$\begin{aligned} Fz_{eg22b} &= -\frac{S_2 V_2^2}{(\pi/2 - \alpha)^2} \int_{x_9}^{x_{10}} \int_{y_9}^{y_{10}} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .dxdy \\ Tx_{eg22b} &= \frac{S_2 V_2^2}{(\pi/2 - \alpha)^2} \int_{x_9}^{x_{10}} \int_{y_9}^{y_{10}} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .ydxdy \end{aligned} \quad (18)$$

Similarly the electrostatic force and torque caused by sidewall 12 on the gimbal frame are

$$\begin{aligned} Fz_{eg12} &= -\frac{S_2 V_1^2}{(\pi/2 + \alpha)^2} \int_{x_{11}}^{x_{12}} \int_{y_{11}}^{y_{12}} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .dxdy \\ Tx_{eg12} &= -\frac{S_2 V_1^2}{(\pi/2 + \alpha)^2} \int_{x_{11}}^{x_{12}} \int_{y_{11}}^{y_{12}} \left\{ \frac{1}{y \cos \alpha - \frac{L_e}{2} - t_s} \right\}^2 .ydxdy \end{aligned} \quad (19)$$

**3.1.2 Bottom Electrodes** The electrostatic forces and torques caused by bottom electrodes should also be added to the simulations. For the rotation angle  $\alpha$ , as it is shown in Figure 5, the flux can be obtained from

$$E_b = \frac{V}{\widehat{AB}} \quad (20)$$

where

$$\begin{aligned} \widehat{AB} &= AC \cdot (\alpha) = \frac{AD}{\sin(\alpha)} (\alpha) \\ &= (g + Y \tan \alpha) \frac{\alpha}{\sin(\alpha)} \end{aligned} \quad (21)$$

Substituting  $Y = y \cos \alpha$ , and using the similar procedure as before, the electrostatic force and torque caused by voltages on quadrants 1 or 3 is derived

$$\begin{aligned} S_3 &= \frac{\epsilon_0 \cdot V_1^2 \sin^2(\alpha)}{2 \alpha^2} \\ Fz_{eb1a} &= -S_3 \cdot \int_{x_{13}}^{x_{14}} \int_{y_{13}}^{y_{14}} \left( \frac{1}{g + y \cdot \sin(\alpha)} \right)^2 .dxdy \\ Tx_{eb1} &= -S_3 \cdot \int_{x_{13}}^{x_{14}} \int_{y_{13}}^{y_{14}} \left( \frac{1}{g + y \cdot \sin(\alpha)} \right)^2 .ydxdy \end{aligned} \quad (22)$$

similarly, the electrostatic force and torque caused by bottom electrodes of quadrants 2 or 4 are

$$\begin{aligned} S_4 &= \frac{\epsilon_0 \cdot V_2^2 \sin^2(\alpha)}{2 \alpha^2} \\ Fz_{eb2a} &= -S_3 \cdot \int_{x_{13}}^{x_{14}} \int_{y_{13}}^{y_{14}} \left( \frac{1}{g + y \cdot \sin(\alpha)} \right)^2 .dxdy \\ Tx_{eb2} &= S_3 \cdot \int_{x_{13}}^{x_{14}} \int_{y_{13}}^{y_{14}} \left( \frac{1}{g + y \cdot \sin(\alpha)} \right)^2 .ydxdy \end{aligned} \quad (23)$$

For rotation angle  $\beta$ , the equations are derived in a similar fashion, except that the forces on the gimbal frame do not contribute to the rotation angle  $\beta$ , and therefore are not considered (Appendix A).

### 3.2 Mechanical Spring Force

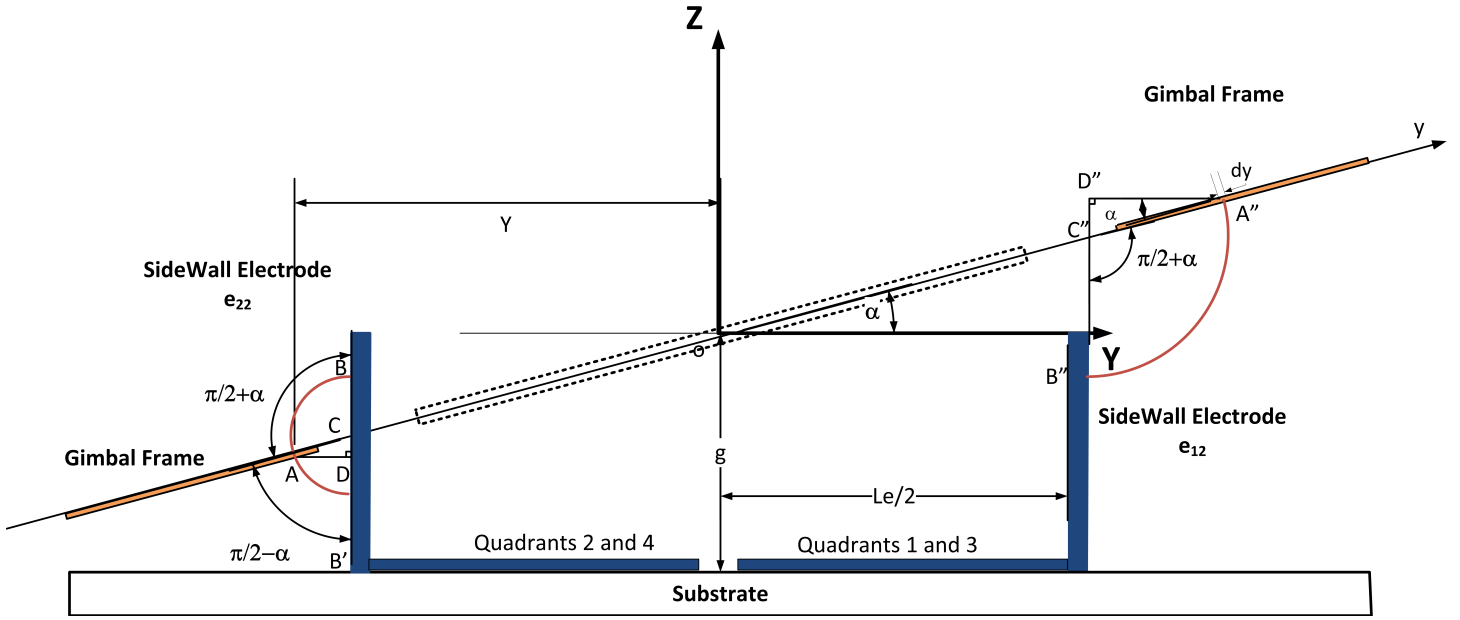
When the micro-mirror is actuated by sidewall electrodes, the electrostatic forces and torques will be balanced by mechanical restoring forces and torques.

$$2 \cdot K_\alpha \alpha = 2(Tx_{e22t} + Tx_{e22b} + Tx_{e12} + (Tx_{eg22t} + Tx_{eg22b}) + Tx_{eg12} + \sum_{m=1}^2 Tx_{ebm})$$

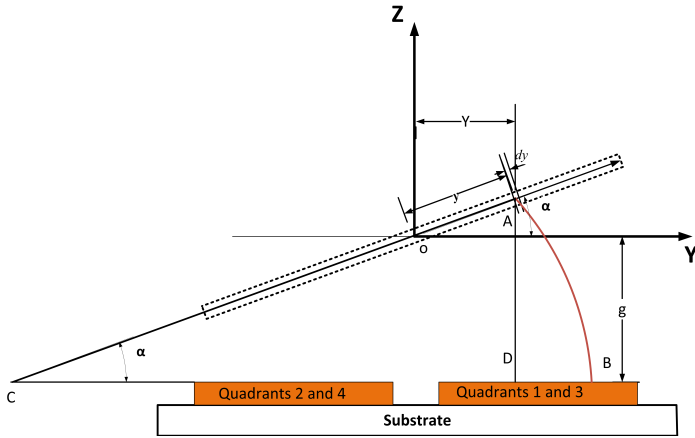
$$2 \cdot K_\beta \beta = 2(Ty_{e13t} + Tx_{e13b} + Ty_{e33} + Ty_{eb1} + Ty_{eb3})$$

$$\begin{aligned} K_z Z &= 2(Fz_{e12} + Fz_{e22b} + Fz_{e22t} + Fz_{ge12} + Fz_{ge22b} + Fz_{ge22t} \\ &\quad + Fz_{eb1a} + Fz_{eb2a} + Fz_{e33} + Fz_{e13b} + Fz_{e13t} + Fz_{eb1b} + Fz_{eb3b}) \end{aligned} \quad (24)$$

In the above equations, the stiffness coefficients are doubled to account for the two springs resisting the motion about the two



**FIGURE 4:** The Projection of the mirror plate, and gimbal and Sidewall electrodes on the YZ Plane



**FIGURE 5:** Bottom electrode effect for  $\alpha$  scanning angle

axes. The serpentine spring has the stiffness [30] of

$$K_{\alpha} = K_{\beta} = \left( \frac{l_f + \frac{6}{4}l_p + \frac{2l_i}{4}}{G \cdot J_r} + \frac{2l_0}{EI} \right)^{-1} \quad (25)$$

where for either of rotation angles, polar moment of inertia  $J_r$ , and moment of inertia of the spring rectangular cross section

$I$  are

$$J_r = \frac{t_b^3 \cdot w}{3} \cdot \left( 1 - \frac{192t_b}{w \cdot \pi^5} \cdot \tanh\left(\frac{\pi \cdot w}{2t_b}\right) \right) \quad (26)$$

$$I = \frac{w \cdot t_b^3}{12}$$

For the simulations, displacement in the z directions are not illustrated and only the rotation angles are simulated using the equations 24 as described in the next section.

#### 4 SIMULATION RESULTS

For obtaining the rotation angles, the summation of all the torques in corresponding directions of x and y are found considering the voltages of each quadrant. Equations 24 were solved at different differential voltages as indicated in equations 1 and 2. The displacement in the z direction has not been presented as the experimental data was not available for comparison. In the simulations, the fringe field effect is also considered negligible. Dimensions and material properties for the simulations, are listed in Table 1.

Figure 6 shows the comparison of the rotation angle  $\alpha$  versus differential voltage based on the simplified analytical model and the experimental and simulation results [30].  $V_{bias}$  is applied as 55V and  $V_{diff}$  ranges from 0-150V. It is noted the simplified

model reveals close results for  $\alpha$  scanning angle in Figure 6 compared with the reported results. However, unlike experiments, our model does not respond nonlinearly to the applied voltage at large angles. Nonlinearity in the response can be due to the gap decrease as the voltage increases. In our simulation, we assumed the gap is constant. More accurate model should account for changing of the gap as the voltage increases and using the calculated gap in the simulated rotation angles. This approach should make the curve nonlinear and similar to a regular pull-in curve.

Figure 7 shows the rotation angle  $\beta$  versus differential voltage when bias voltage  $v_{bia} = 55V$ . It can be seen that the simplified model underestimate the scanning angles. For both rotation angles, we used the same mechanical stiffness for the springs and that can be the cause for deviation seen in the simulation of the rotation angle  $\beta$ . It is more likely that the mirror experiences more resistance in rotation about the X axis compared to the rotation about the Y axis. Overall, the model predicts the rotation angle  $\alpha$  with more accuracy compared to the rotation angle  $\beta$ , which is underestimated.

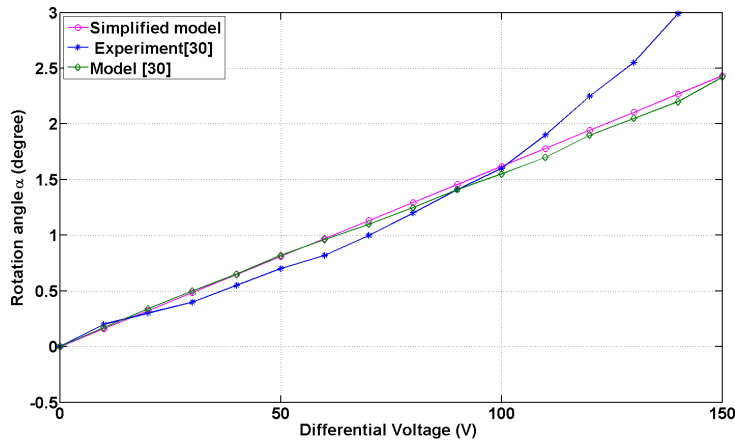


FIGURE 6: Alpha Scanning Angle when the  $V_{bias} = 55V$

## 5 CONCLUSIONS

An analytical model for a bi-axial micro-mirror with side-wall electrodes is presented, which is a simplified model of a previous study [30] based on independent excitation of the two rotation angles. The mirror rotates about X and Y axes as a result electrostatic torques from different voltages on the mirror plate, side-wall electrodes, gimbal frame and the bottom electrodes. Actuation of the mirror is done using differential voltage excitation for the four quadrants and therefore, the rotations about axes are done independently which enabled simplifying the

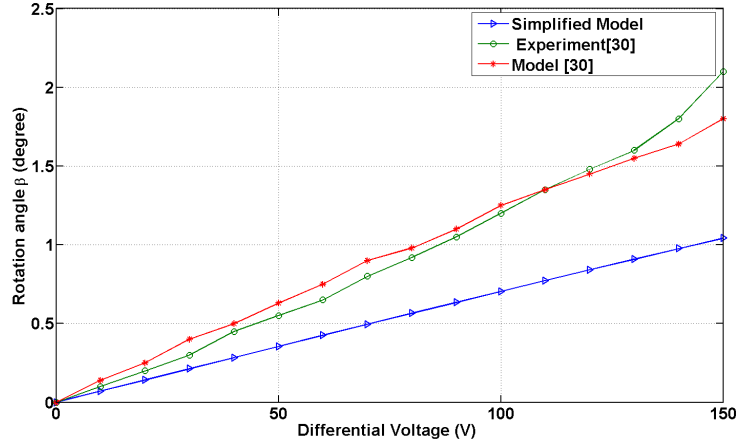


FIGURE 7: Beta Scanning Angle when the  $V_{bias} = 55V$

equations. The integral boundaries are also chosen such that they account for forces applied on the mirror as the rotation angle increases. The simulation results of rotation angle about the X axis are in close proximity of the experiments, while simulation for rotation about Y axis underestimate the angles. Our future work accounts for the stiffness differences for rotations about the two axes. The simplified model with less computation time enables adding optimization algorithm, which was the motivation of this study to reduce the operating voltage and to increase the rotation angles.

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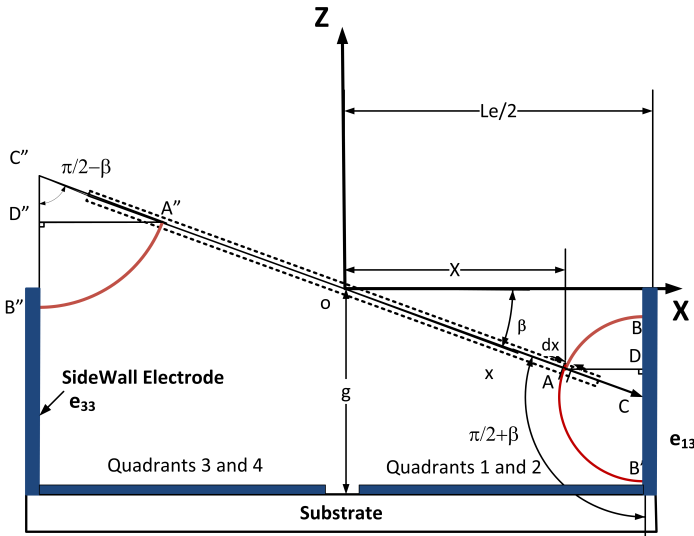
### Appendix A: Equations for rotation angle $\beta$

Sidewall 13 creates electrostatic forces on the top and bottom parts of the mirror plate. However, similar to rotation angle  $\alpha$ , the top forces only appear when the rotation angle  $\beta \geq 0.06$  radians. For an element of  $dx$  on the top layer of the mirror plate (8), the electrostatic flux from sidewall 13 is written as;

$$E_{e13t} = \frac{V}{\widehat{AB}} \quad (27)$$

where

$$\begin{aligned} \widehat{AB} &= AC \cdot (\pi/2 - \beta) = \frac{AD}{\sin(\pi/2 - \beta)} (\pi/2 - \beta) \\ &= \left(\frac{L_e}{2} - X\right) \frac{\pi/2 - \beta}{\sin(\pi/2 - \beta)} \end{aligned} \quad (28)$$



**FIGURE 8:** The mirror rotated about Y axis by  $\beta$ , and the sidewalls 13 and 33 projected on the XZ plane.

Substituting  $X = x \cos \beta$  for a body fixed point A, the flux equation 27 is written as;

$$E_{e13t} = \frac{V \cdot \sin(\pi/2 - \beta)}{\left(\frac{L_e}{2} - x \cdot \cos \beta\right) (\pi/2 - \beta)} \quad (29)$$

The electrostatic pressure is

$$P = \frac{\epsilon_0 E^2}{2} \quad (30)$$

Therefore the electrostatic force on an element with dimensions  $dx$  and  $dy$  is

$$dF = \frac{\epsilon_0 E^2 dx dy}{2} \quad (31)$$

The electrostatic force in the Z direction and the torque about the Y axis from sidewall e13 on top surface of the mirror plate are then obtained.

$$\begin{aligned} S_5 &= \frac{\epsilon_0 \cdot \sin^2(\pi/2 - \beta)}{2} \\ F_{ze13t} &= \frac{S_5 V_1^2}{(\pi/2 - \beta)^2} \cdot \int_{y_{15}}^{y_{16}} \int_{x_{15}}^{x_{16}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot dx dy \\ T_{ye13t} &= -\frac{S_5 V_1^2}{(\pi/2 - \beta)^2} \cdot \int_{y_{15}}^{y_{16}} \int_{x_{15}}^{x_{16}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot x dx dy \end{aligned} \quad (32)$$

For an element of  $dx$  at the bottom surface of the mirror plate (8), the electrostatic flux from sidewall 13 is written as;

$$E_{e13b} = \frac{V_1}{\widehat{AB}'} \quad (33)$$

where

$$\begin{aligned} \widehat{AB}' &= AC \cdot (\pi/2 + \beta) = \frac{AD}{\sin(\pi/2 - \beta)} (\pi/2 + \beta) \\ &= \left(\frac{L_e}{2} - X\right) \frac{\pi/2 + \beta}{\sin(\pi/2 - \beta)} \end{aligned} \quad (34)$$

Substituting  $X = x \cos \beta$  for a body fixed point, the flux equation (33) is written as;

$$E_{e13b} = \frac{V_1 \sin(\pi/2 - \beta)}{\left(\frac{L_e}{2} - x \cdot \cos \beta\right) (\pi/2 + \beta)} \quad (35)$$

The force and torque equations are then derived

$$F_{ze13b} = -\frac{S_5 V_1^2}{(\pi/2 + \beta)^2} \cdot \int_{y_{17}}^{y_{18}} \int_{x_{17}}^{x_{18}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot dx dy$$

$$T_{ye13b} = \frac{S_5 V_1^2}{(\pi/2 + \beta)^2} \cdot \int_{y_{17}}^{y_{18}} \int_{x_{17}}^{x_{18}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot x dx dy$$
(36)

Electrostatic forces from sidewall 33 also changes the rotation angle  $\beta$ . The force and torque on the mirror plate from sidewall 33 can be derived similarly to equation (32) with different boundary for the integral.

$$F_{ze33} = -\frac{S_5 V_3^2}{(\pi/2 - \beta)^2} \cdot \int_{y_{19}}^{y_{20}} \int_{x_{19}}^{x_{20}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot dx dy$$

$$T_{ye33} = -\frac{S_5 V_3^2}{(\pi/2 - \beta)^2} \cdot \int_{y_{19}}^{y_{20}} \int_{x_{19}}^{x_{20}} \left\{ \frac{1}{\frac{L_e}{2} - x \cdot \cos \beta} \right\}^2 \cdot x dx dy$$
(37)

The forces and torques generated by bottom electrode of quadrants 1 or 2 to change  $\beta$  angle (9) are

$$S_6 = \frac{\epsilon_0 \cdot V_1^2 \sin^2(\beta)}{2 \beta^2}$$

$$T_{yeb1} = S_6 \cdot \int_{y_{21}}^{y_{22}} \int_{x_{21}}^{x_{22}} \left( \frac{1}{g - x \cdot \sin(\beta)} \right)^2 \cdot x dx dy$$

$$F_{zeb1b} = -S_6 \cdot \int_{y_{21}}^{y_{22}} \int_{x_{21}}^{x_{22}} \left( \frac{1}{g - x \cdot \sin(\beta)} \right)^2 \cdot dx dy$$
(38)

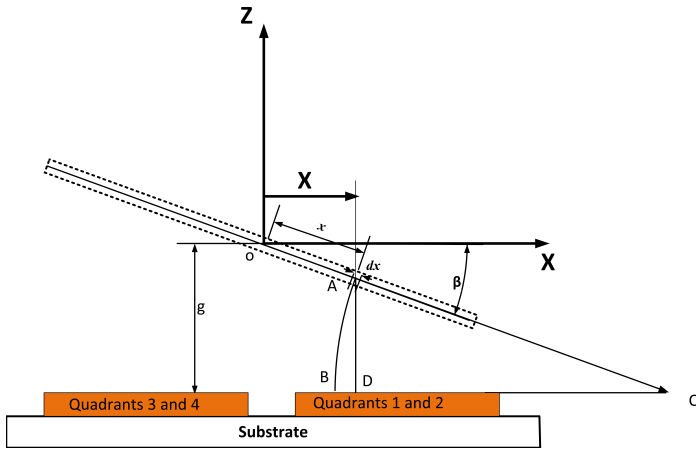


FIGURE 9: Bottom electrode effect for  $\beta$  scanning angle

## Appendix B: Integral Boundaries

$$y_1 = \frac{Le}{2 \cos(\alpha_0)} - \frac{Le}{2} \tan(\alpha_0) \quad y_2 = \frac{lm}{2}$$

$$x_1 = \frac{w_f}{2} \quad x_2 = \frac{lm}{2}$$

$$y_3 = \frac{Le}{2 \cos(\alpha_0)} - g + \frac{Le}{2} \tan(\alpha_0) \quad y_4 = \frac{lm}{2}$$

$$x_3 = \frac{w_f}{2} \quad x_4 = \frac{lm}{2}$$

$$y_5 = \frac{w_f}{2} \quad y_6 = \frac{lm}{2}$$

$$x_5 = \frac{w_f}{2} \quad x_6 = \frac{lm}{2}$$

$$y_7 = \frac{Lgl}{2} - g_{wy} \quad y_8 = \frac{Le}{2 \cos(\alpha_0)} + \frac{Le}{2} \tan(\alpha_0)$$

$$x_7 = \frac{w_f}{2} \quad x_8 = \frac{Le}{2} + t_s$$

$$y_9 = \frac{Lgl}{2} - g_{wy} \quad y_{10} = \frac{Le}{2 \cos(\alpha_0)} + g - \frac{Le}{2} \tan(\alpha_0)$$

$$x_9 = \frac{w_f}{2} \quad x_{10} = \frac{Le}{2} + t_s$$

$$y_{11} = \frac{Lgl}{2} - g_{wy} \quad y_{12} = \frac{Le}{2 \cos(\alpha_0)} + \frac{Le}{2} \tan(\alpha_0) + g$$

$$x_{11} = \frac{w_f}{2} \quad x_{12} = \frac{Le}{2} + t_s$$

$$y_{13} = \frac{w_f}{2} \quad y_{14} = \frac{lm}{2}$$

$$x_{13} = \frac{w_f}{2} \quad x_{14} = \frac{lm}{2}$$

$$x_{15} = \frac{Le}{2 \cos(\alpha_0)} - \frac{Le}{2} \tan(\alpha_0) \quad x_{16} = \frac{lm}{2}$$

$$y_{15} = \frac{w_f}{2} \quad y_{16} = \frac{lm}{2}$$

$$x_{17} = \frac{-Le}{2} + g \quad x_{18} = \frac{lm}{2}$$

$$y_{17} = \frac{w_f}{2} \quad y_{18} = \frac{lm}{2}$$

$$x_{19} = \frac{Le}{2} - g \quad x_{20} = \frac{lm}{2}$$

$$y_{19} = \frac{w_f}{2} \quad y_{20} = \frac{lm}{2}$$

$$x_{21} = \frac{w_f}{2} \quad x_{22} = \frac{lm}{2}$$

$$y_{21} = \frac{w_f}{2} \quad y_{22} = \frac{lm}{2}$$